

Seismic Control of Asymmetric Structures

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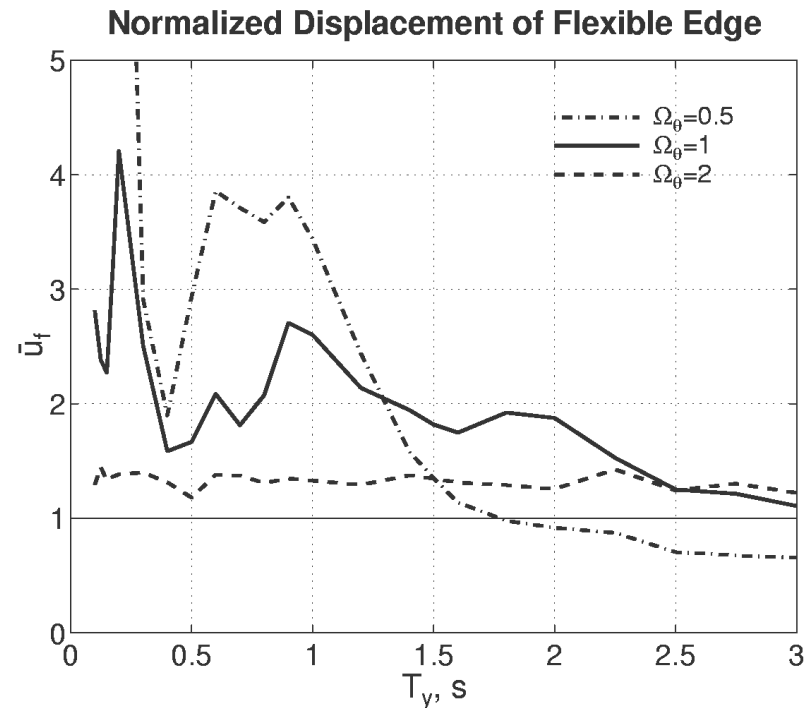


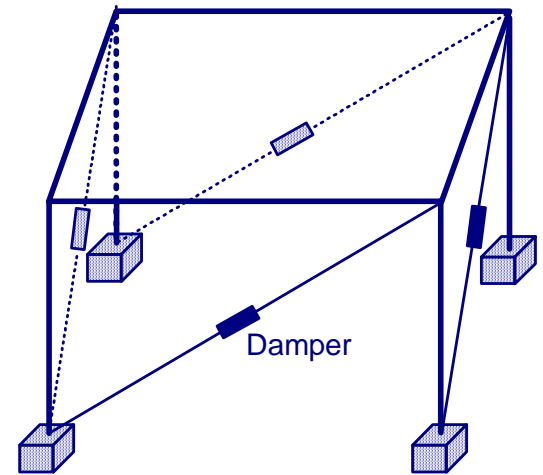
Objectives

- Systematically investigate effects of viscous supplemental damping on response of asymmetric structures
 - ➔ Identify controlling system parameters
 - ➔ Study effects on deformations
 - ➔ Develop an understanding of key parameters controlling the response
 - ➔ Develop simplified procedure for distributing supplemental damping
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Motivation: Excessive edge deformation in asymmetric structures


- Excessive damage in edge structural/nonstructural elements
- Increased potential for pounding between adjacent buildings
- Higher second order (P- Δ) effects








System parameters without supplemental damping

- Lateral period of vibration, T_y
 - Stiffness eccentricity \bar{e}
 - Torsional to lateral frequency ratio, Ω_θ
 - Structural damping ratio, ζ
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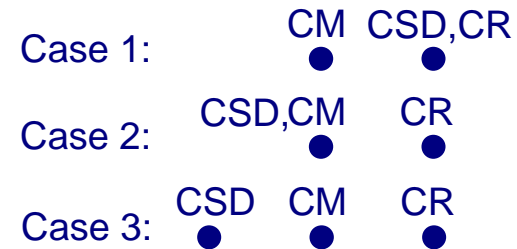


Additional parameters to characterize supplemental damping

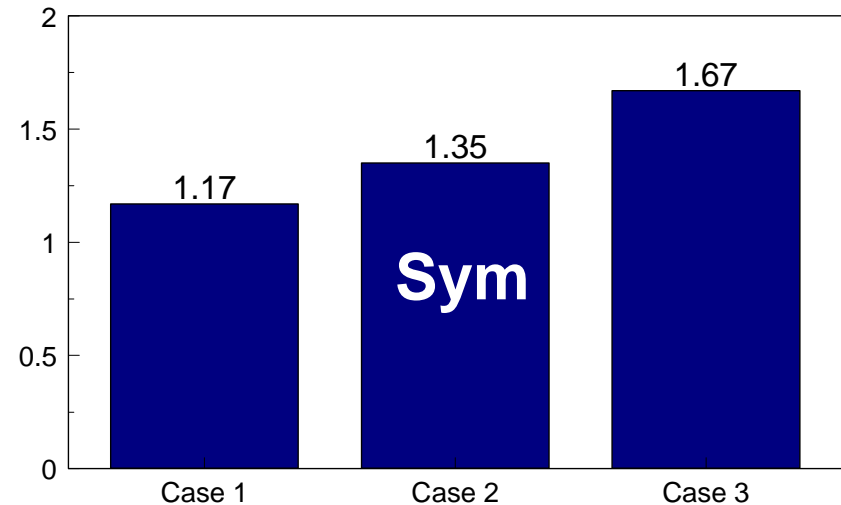
- Supplemental damping ratio, ζ_{sd}
 - Additional damping due to supplemental devices
 - Damping eccentricity \bar{e}_{sd}
 - Location of Center of Supplemental Damping with respect to Center of Mass
 - Damping radius of gyration \bar{r}_{sd}
 - Plan-wise spread of supplemental devices
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Reduction in edge deformation due to asymmetric damping

- Asymmetric distribution may give higher reduction
 - case 3 Vs case 2
- Not all asymmetric distributions give higher reduction
 - case 1 Vs case 2



Reduction Factor = Resp. Without / Resp. With





Summary of results from a parametric study

- Supplemental damping reduces edge deformations
 - ➔ Higher reduction with higher damping
- Degree of reduction depends on the plan-wise distribution of damping
 - ➔ Supplemental damping eccentricity (Strong)
 - ➔ Supplemental damping radius of gyration (Weak)





Quadratic eigen-value problem (State-Space formulation)

$$(\mathbf{B} + \lambda \mathbf{A}) \Phi = \mathbf{0}$$

$$\mathbf{A} = \begin{bmatrix} -\mathbf{K} & \mathbf{0} \\ \mathbf{0} & \mathbf{M} \end{bmatrix} \quad \mathbf{B} = \begin{bmatrix} -\mathbf{0} & \mathbf{K} \\ \mathbf{K} & \mathbf{C} \end{bmatrix}$$

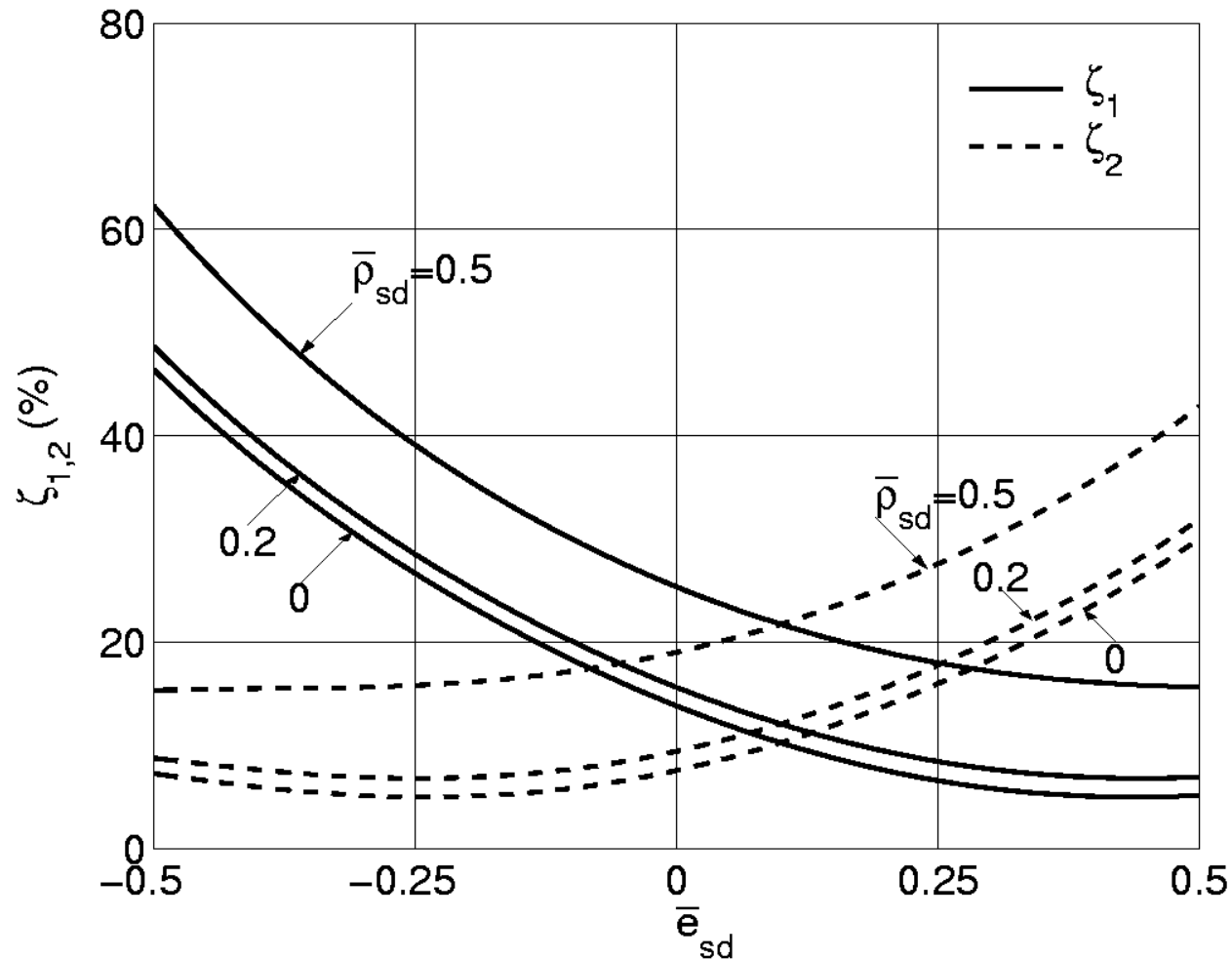
$$\lambda_n = \zeta_n \omega_n \mp j \omega_n \sqrt{1 - \zeta_n^2}$$

ζ_n = Apparent Modal Damping

ω_n = Apparent Modal Frequency



Variation of apparent modal damping





Steady-state response to harmonic loading

$$z_{kn} = \frac{2R_{dn}}{\omega_n} C_{kn} \Gamma_n \Phi_{kn} \ddot{u}_{go}$$

R_{dn} = Dynamic Response (Amplification Factor)

Γ_n = Modal Participation Factor

Φ_{kn} = Mode Shape Component

C_{kn} = Constant Depending on Phase Angles





Effect of supplemental damping distribution

- Quantities not affected significantly

Γ_n = Modal Participation Factor

Φ_{kn} = Mode Shape Component

C_{kn} = Constant Depending on Phase Angles

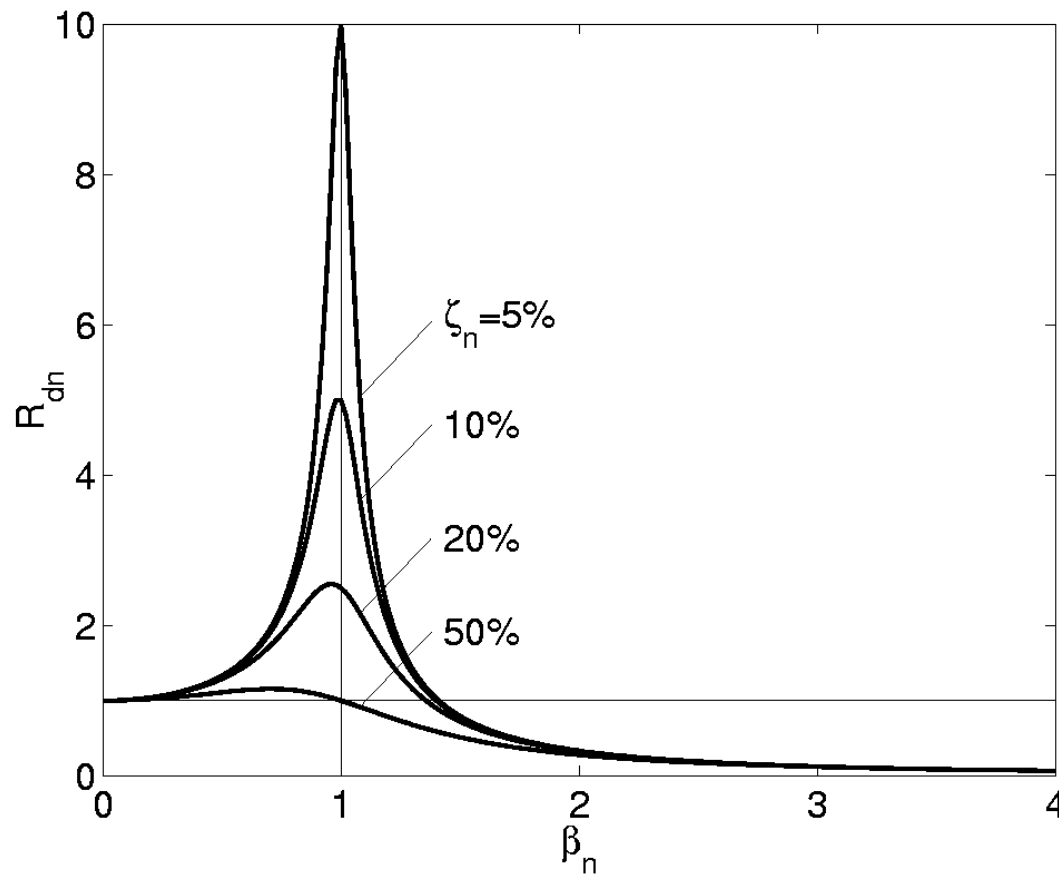
ω_n = Apparent Modal Frequency

- Quantity affected significantly

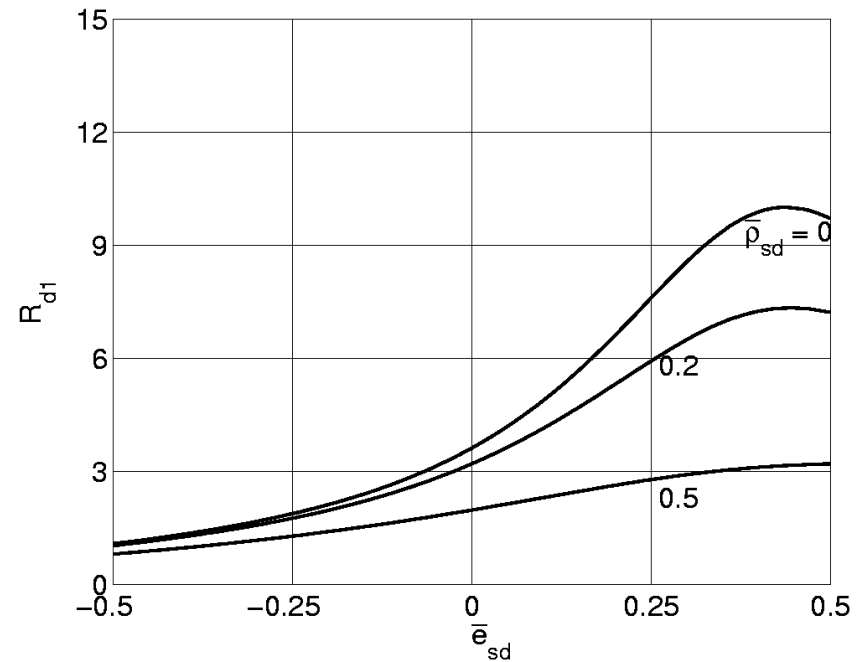
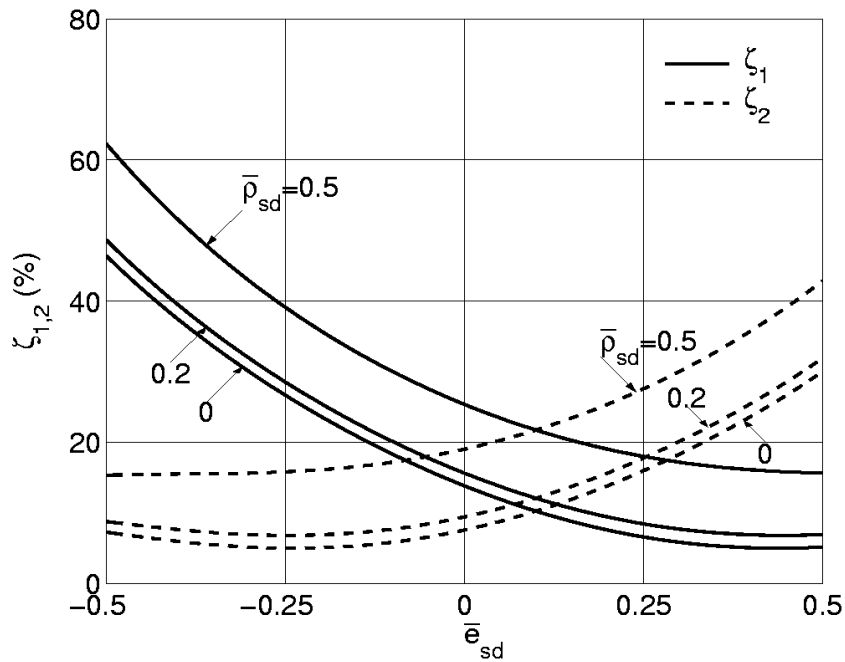
R_{dn} = Dynamic Response (Amplification Factor)



Variation of dynamic response factor with damping

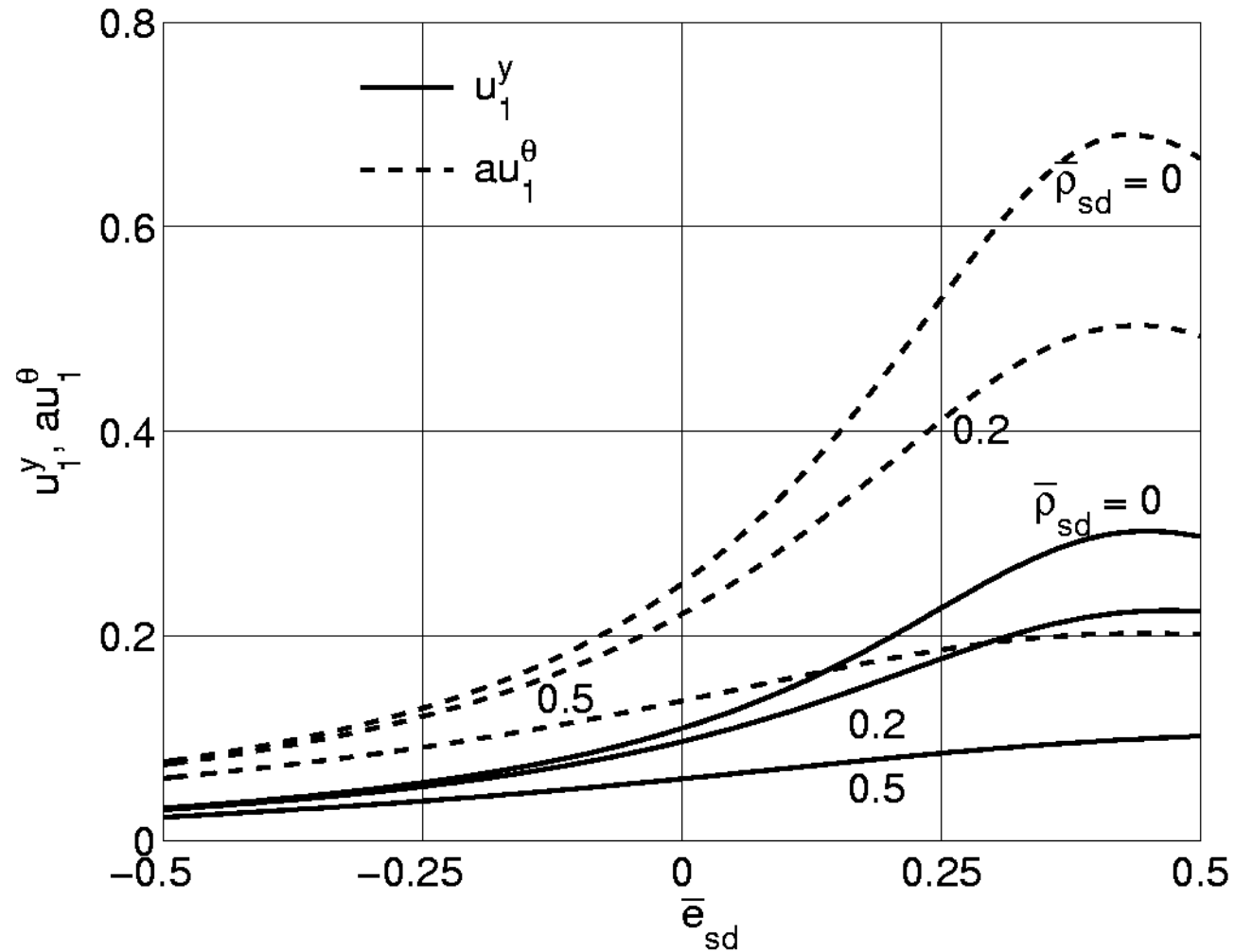


Effect of apparent modal damping on R_{dn} ($\beta_1 = \omega/\omega_1 = 1$)

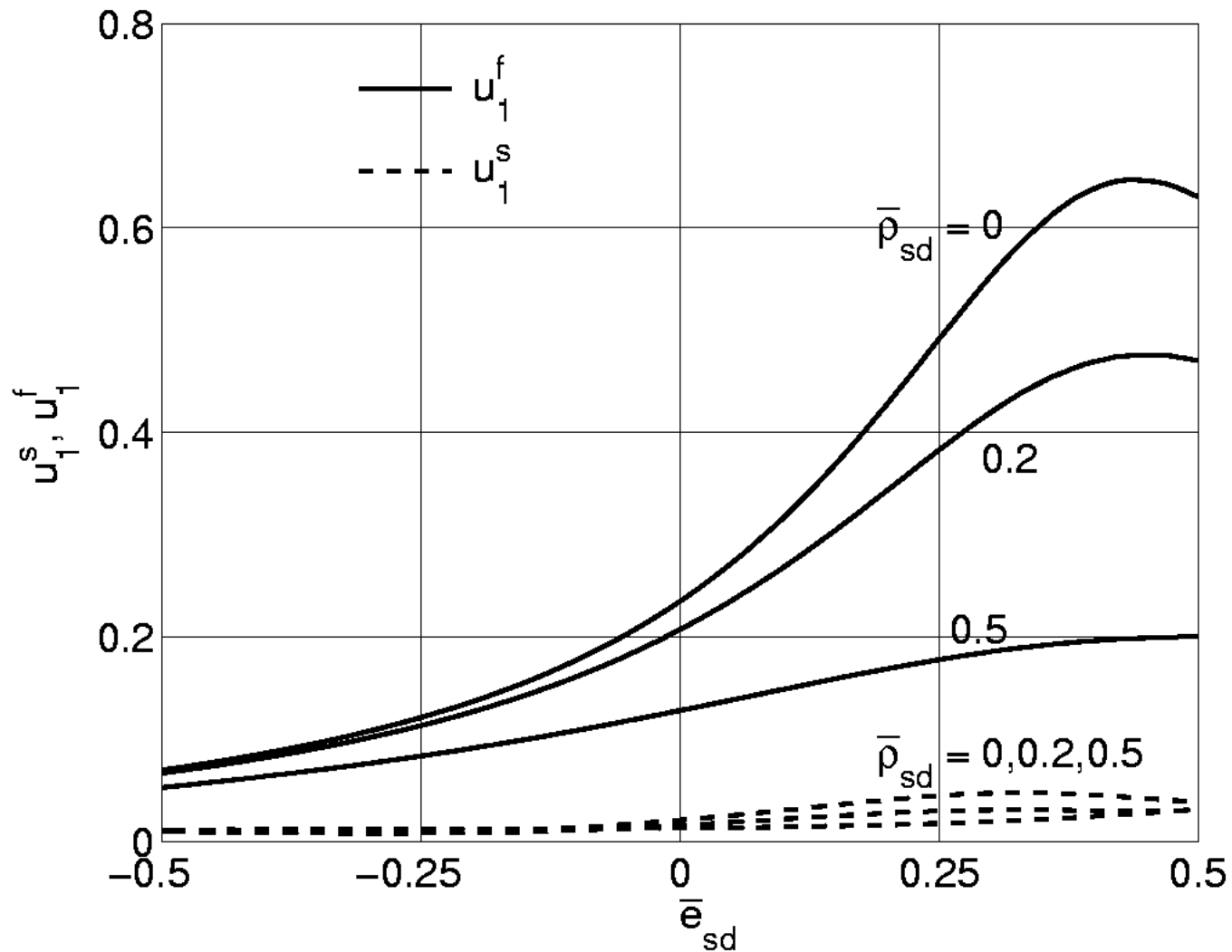




Variation of deformations at CM



Variation of edge deformation



Conclusions

- Larger reduction with asymmetric distribution of supplemental damping
- CSD on opposite side of CM as CR for controlling deformations (at CM as well as at edges)
- Apparent modal damping is the key parameter that controls the response
 - ➔ Simplified procedures should be able to correctly capture the apparent modal damping